# The Pennsylvania State University Steam Services Building



# **Final Thesis Report**

AE 482: Architectural Engineering Thesis April 22, 2019 Mitchell Seltzer, Mechanical Option Dr. William Bahnfleth, Faculty Advisor







Contractor: Alexander Building Construction Co Architect: Buchart Horn Architects Civil: Buchart Horn Architects Structural: Buchart Horn Architects MEP: Buchart Horn Architects Interior: Diversified Design Owner: The Pennsylvania State University

#### Construction

- 4 story, 33,000 square feet mixed-use building
- Features machine and shop rooms as well as office spaces, which requires significant noise reduction features.
- Modern design featuring precast stone and modified flemish bond brick facade. The facilities design is intended to complement the existing west campus steam plant and meant to tie into surrounding campus designs.
- 25 foot vertically spanning curtain wall windows on the east, south and west sides. Cantilevered design atop the all glass main lobby featuring floor to ceiling curtain walls on the first floor.

#### MEP

- One rooftop DOAS with an energy recovery unit, vertical fan coil units, blower coil units all tied into campus steam heat and cooling loops.
- A mini-split system with a condensing outside at ground level.
- Main electrical transformer ties into the existing steam plant providing a high voltage of 480/277v and a low voltage of 208/120v 3-phase, with distribution panels on each floor and different circuits each room.
- Emergency generator backup for egress and a wet pipe sprinkler system serving all floors.

#### Structural

- Concrete spread footings with CMU foundation walls on the south half.
- Concrete basement walls on the north half of the building with a concrete slab on grade.
- Steel superstructure with webbed floor joists.

#### Mitchell Seltzer | Mechanical

#### **Steam Services Building**

Dr. William Bahnfleth https://mps5404.wixsite.com/aethesis State College, PA

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### **Executive Summary**

The Pennsylvania State University New Steam Services Building At The West Campus Steam Plant is a 33,000 square foot multipurpose office building. The building sits within the West Campus Steam Plant's courtyard which previously housed the coal fired boilers, bag house and smoke stack. In 2016 the West Campus Steam Plant begun its transition from a coal fired system to a exclusively natural gas based system. After the plant's conversion to natural gas the university was able to demolish the boilers, bag house and smoke stack, providing the lot in which the New Steam Services Building is being constructed on.

The facility is being constructed by Alexander Building Construction Co. for the Energy Services division at Penn State University. The four story building features a spread footing foundation, a steel structure, and a decentralized HVAC system. The design and construction of the facility is extremely important to the Energy Services department as the building will largely serve as the center for students and visitors to understand the advanced operations that occur at the campus steam plants.

Within the multipurpose office building, the basement and North section of the first floor feature industrial shops spaces. These spaces will be used by the Penn State Steam Plant workers to fabricate, repair and assemble equipment that will be used next door in the West Campus Steam Plant. The second floor and the South portion of the first floor consist of office spaces as well as a break room, a training room, and storage spaces. A small section of the second floor and most of the third floor is currently shell space for future expansions.

The decentralized HVAC system features a dedicated outdoor air system with an energy recovery wheel. The system also features a split system for the telecom room. The building's heat is supplied by a steam line to shell and tube heat exchangers, and the heating and cooling is distributed through a four pipe heating and cooling system.

In the following report there will be two mechanical redesign proposals. First, active chilled beams will be installed, while utilizing the existing dedicated outdoor air system equipment. Next will be the Air Handling Unit Changeover-Bypass System with variable air volume boxes. Here, the dedicated outdoor air unit will be updated to the new air handling unit, and spaces served by fan coil units with the variable air volume boxes. Both proposals will include an analysis for cost, payback, and energy savings. In addition to the mechanical redesign proposals, there will also be two breadth topics that analyze the effects of the proposed air handling unit on both the structural and electrical systems.

### **Existing Conditions**

The Penn State Steam Services Building currently has a mechanical system consisting of a decentralized HVAC system with a dedicated outdoor air system, which includes an energy recovery wheel. This system also includes a split system for the telecom room. Building side heating is performed via steam. The system is served by The Pennsylvania State University West Campus Steam Plant building through a steam line that goes through shell and tube heat exchangers that supplies the building through a hot water loop. This is connected to a series of 46 fan coil units, 2 blower coil units, 3 unit heaters, and a dedicated outdoor air system with hydronic coils. Refer to the image below for the Hot Water Diagram.

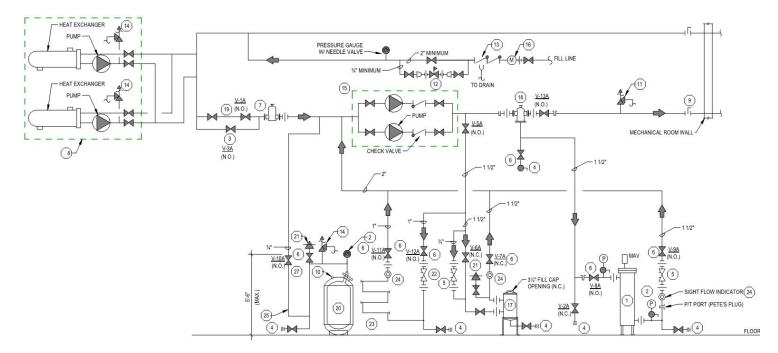


Figure 1: Hot Water Riser Diagram

Building side cooling is performed via chilled water. The system is served by The Pennsylvania State University West Campus Chiller building. Chilled water is supplied via a 3" Supply and Return line and pumped via Chilled Water Pumps from the pump room in the basement to a chilled water loop that is connected to a series of 46 fan coil units, 2 blower coil units, and a dedicated outdoor air system with hydronic coils. Refer to the image below for the Chilled Water Diagram.

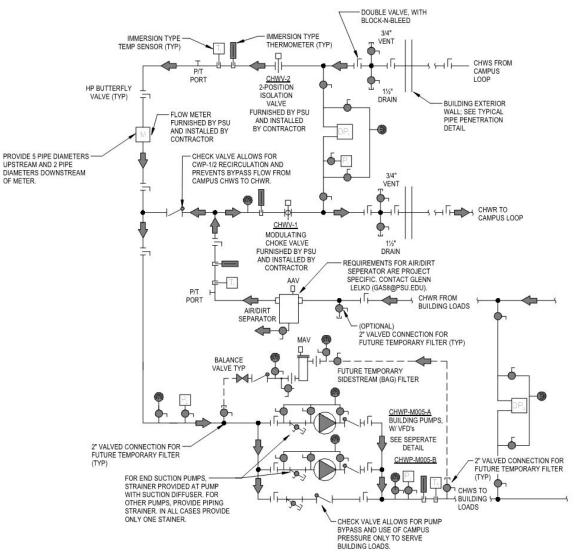


Figure 2: Chilled Water Riser Diagram

Building loads were found by performing an energy model of the building on the software Trane Trace. The following tables outline the building energy use, utility use, and building operation cost. These values will be the baseline used to determine the feasibility of the proposed mechanical designs.

Table	1: Total	Energy	Use	per	Year
IUNIC	<b>1</b>		0.00	PC:	

Total Building Energy per Year (kBtu/yr)	6,458,112.00
Total Source Energy per Year (kBtu/yr)	12,619,165.00

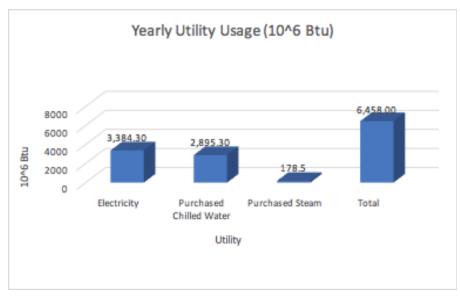


Figure 3: Yearly Utility Usage

	Electric (kWh)	Steam (therms)	Chilled Water (therms)
January	82,938	390	2,269
February	74,896	344	2,058
March	82,878	297	2,489
April	80,195	173	2,578
May	83,002	62	2,099
June	80,459	59	2,588
July	83,105	71	3,159
August	83,245	66	2,206
September	80,386	34	2,079
October	82,918	91	2,346
November	80,195	164	2,514
December	82,879	324	2,340

## Table 2: Monthly Utility Usage

Energy Cost per Year	\$ 97,633.00	
Initial Cost	\$ 215,000.00	
Maintenance Cost per Year	\$ 6,363.65	
15 Year Life Cycle Cost	\$ 1,774,949.00	

### Table 3: Energy and Life Cycle Cost

## **Active Chilled Beams**

To keep the existing air-water system of the current mechanical system, active chilled beams were chosen as the first mechanical redesign proposal. Active chilled beams work by the two thermodynamic properties of radiation and convection, and provide heating and cooling to spaces with necessary additional ventilation from the air handling unit.

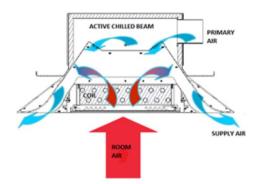


Figure 4: Active Chilled Beam Schematic

To calculate the number and size of the active chilled beams required for the building, Trane Trace was first used to find the total load of the chilled beams. Using their load selection, the loads for each room were determined and entered into Titus Teams's Active Chilled Beams selection software. In addition to these the latent load, sensible load, heating load, and airflow for each room was entered. Once the software was run it yielded CBAL-12 type active chilled beams at 2', 4', 6', and 8'.

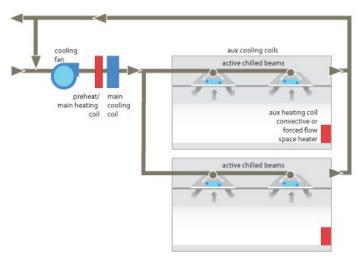


Figure 5: Active Chilled Beam System Schematic

Active Chilled Beam				
	First Cost			
Size	Cost	Number	Total	
8'	950	41	\$ 38,950	
6'	750	49	\$ 36,750	
4'	650	6	\$ 3,900	
2'	500	29	\$ 14,500	
TOTAL		125	\$ 94,100	

**Table 4: Active Chilled Beam Software Results** 

Table 5: Energy Use per Year

Total Building Energy per Year (kBtu/yr)	6,680,735.00
Total Source Energy per Year (kBtu/yr)	12,624,769.00

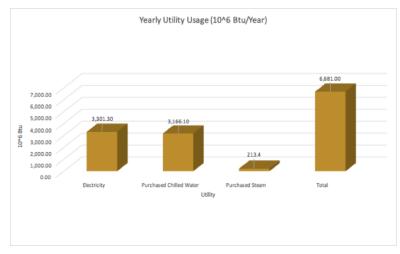


Figure 6: Yearly Utility Use of Active Chilled Beam System

Energy Cost per Year	\$ 96,691.00
Initial Cost	\$ 159,100.00
Maintenance Cost per Year	\$ 1,417.45

#### Table 6: Energy and Life Cycle Cost

15 Year Life Cycle Cost	\$ 1,630,726.75
Savings per Year	\$ 5,888.20

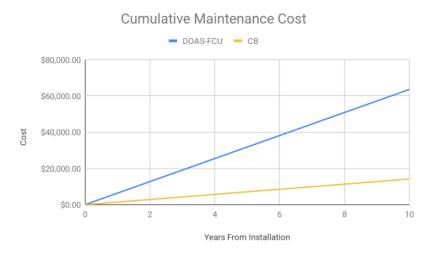


Figure 7: Yearly Progressive Maintenance Cost of Original System versus Active Chilled Beam System

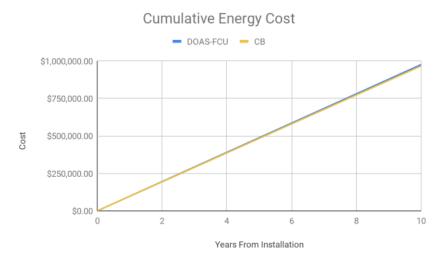


Figure 8: Yearly Progressive Energy Cost of Original System versus Active Chilled Beam System

$$Payback Period = \frac{Initial Cost}{Savings} = \frac{\$159,100}{\$5,888.20} = 27 years$$

The first mechanical redesign proposal will be an air handling unit changeover-bypass system with variable air volume units. Hot water and chilled water will be provided to the rooftop air handling unit from the Penn State campus steam and chiller plants. Included in the air handling unit will be a desiccant energy wheel, used to help with latent and sensible recovery for both heating and cooling seasons, standard air filters, and fans that supply both necessary ventilation air and enough for heating and cooling. Air will be selectively distributed throughout the building with the use of variable air volume (VAV) units, which controls the volume of supply air to zones based on zone loads. In the occasion that the VAV zones do not require the full amount of air supplied by the air handling unit, then the changeover-bypass system will return the excess air back to the air handling unit.

In order to correctly size the proposed air handling unit, Trane Trace was first used to determine the heating and cooling loads of the building under this new system. The company H.C. Nye Co. was then consulted to assist in choosing a model of the correct size.

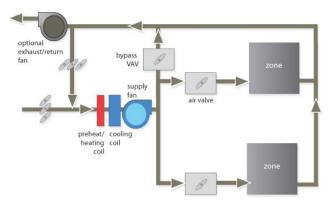


Figure 9: Air Handling Unit with VAV Schematic Diagram

Maximum Air Flow	15,946 CFM	
Cooling - Sensible Capacity	474.45 MBH	
Cooling - Latent Capacity	204.60 MBH	
Heating Capacity	228.7 MBH	

#### **Table 7: Air Handling Unit Specifications**

Total Building Energy per year (kBtu/yr)	5,444,031.00
Total Source Energy per year (kBtu/yr)	11,683,876.00

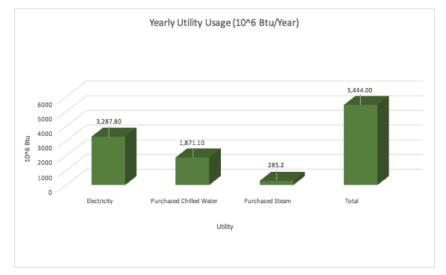


Figure 10: Yearly Utility Use of Air Handling Unit System

Energy Cost per Year	\$ 94,770.00		
Initial Cost	\$ 114,000.00		
Maintenance Cost per Year	\$ 548.20		
15 Year Life Cycle Cost	\$ 1,543,773		
Savings per Year	\$ 8,678.45		



Figure 11: Yearly Progressive Maintenance Cost of Original System versus Air Handling Unit System

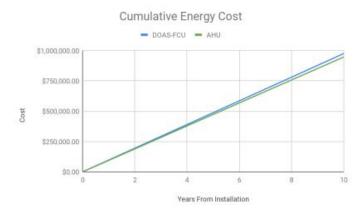


Figure 12: Yearly Progressive Energy Cost of Original System versus Air Handling Unit System

$$Payback Period = \frac{Initial Cost}{Savings} = \frac{\$114,000}{\$8,678.45} = 13 years$$

# **Depth Proposal Comparison**

After completing each mechanical redesign analysis, the final system recommendation was chosen to be the proposed air handling unit with VAV units throughout the building. This system was chosen based off of a number of results from the analysis. Not only does this system have the lower payback period of 13 years, but also has a much lower annual energy use and upfront cost.



Figure 13: Air Handling Unit

Total Building Energy Savings per Year (kBtu/yr)	1,014,081
Total Source Energy Savings per Year (kBtu/yr)	935,289
Initial Cost Savings	\$ 101,000.00
Cost Savings per Year	\$ 8,678.45
15 Year Cost Savings	\$ 231,176.75
25 Year Cost Savings	\$ 317,961.25

#### Table 10: Air Handling Unit with VAV Cumulative Analysis Results

To support the proposed air handling unit on the roof of the existing building, a structural breadth was chosen which consists of resizing the roof support and decking. Curb dimensions and the weight of the proposed air handling unit were supplied by H.C. Nye Co..

**Calculating Weight per Square Foot Allowance:** 

Total Weight = 8,864 lbs Area = 192.94 SF 8,864 lbs /192.94 SF = 45.94 psf => **Weight per SF Allowance = 75 psf** 

The structure of the roof where the equipment will sit consists of girders that are 27'4" center to center, and joists that are 6' 2.5" center to center.

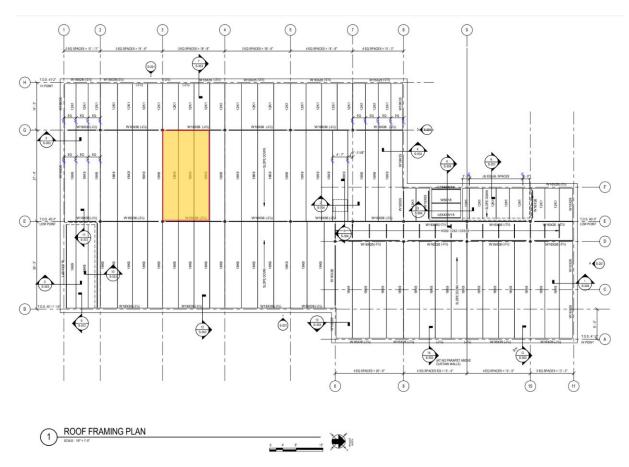


Figure 14: Location of Proposed Air Handling Unit on Roof

**Dead Loads:** Membrane = 1.5 psf Mechanical & Lighting Standard DL = 15 psf

#### Factored Snow Load:

$$\begin{split} P_{f} &= 0.7 * C_{e} * C_{t} * C_{s} * I_{s} * P_{g} \\ C_{e} (Exposure Factor) &= exposure of type C from drawings, and using Table 7-2 from ASCE 7, factor =$$
**0.9** $\\ C_{t} (thermal factor) &= assumed to be$ **1** $\\ C_{s} (slope factor) &= assumed to be$ **1** $\\ I_{s} (importance factor) &= category 2 - factor =$ **1.0** $\\ P_{g} (Snow Load) &= from State College Ordinance at$ **40 psf** $\\ P_{f} &=$ **25.2 psf** $\end{split}$ 

#### (1) Size the Decking for Equipment

Using Vulcraft 1.5B Type Steel Roof Decking at 6' span, 3 spans each

Dead Load + Equipment Load + Snow Factored Load = 16.5 psf + 75 psf + 25.2 psf

#### Allowable Total Load = 116.7 psf

Using Vulcraft 1.5B Type Steel Roof Decking at 6' span, 3 spans each Deck Type **B19** (130 psf) is able to support the Allowable Total Load (116.7 psf) and is the most economical and allowable deck.

# VERTICAL LOADS FOR TYPE 1.5B

No. of Spans	Deck	Max.		Allowable			
	Type SDI Const. Span		5-0	5-6	6-0		
3	B24	5'-10	154 / 120	128 / 90	108 / 69		
	B22	6'-11	124 / 167	103 / 126	87 / 97		
	B20	7'-9	159 / 209	132 / 157	111/121		
	B19	8'-5	186 / 250	154 / 188	130/145		
	B18	9'-1	210 / 289	174 / 217	147 / 167		
	B16	10'-3	264 / 369	219/277	185/214		

Figure 15: Vulcraft Sizing Table for 1.5B Steel Roof Deck

#### (2) Size the Joist for the Equipment

Using Newmill's Economical Load Tables at a 27' joist span

 $LRFD = (1.2 L_D + 1.6 L_L) * ft$ = 1.2(16.5) + 1.6(100.2) = 180.12 psf 180.12(6') = 1080.72 plf

LL(psf) = Equipment Load + Snow Factored Load LL(plf) = LL(psf) \* ft =100.2 \* 6 = 601.2 plf

Using LRFD = 1080.72 plf and LL = 601.2 plf

And Newmill's Economical Load Tables at a 27' joist span

Joist type **20LH08** was chosen due to the live load having a greater factor then the LRFD and being the most economical

Joist Span (ft)		Total Loa	ad (plf)	there be		WEDG	Joist	
	Factored S		Service	Live Lo	ad (plf)	Joist Designation	Weight (plf)	
			ASD	1/240	1/360	Designation		
27 (cont.)	F	768	512	512	406	22K7	8.5	
	F	825	550	550	522	26K7	8.8	
	F	849	566	566	406	20LH04	10.9	
	F	913	609	609	437	20LH05	11.3	
	F	972	648	621	414	18LH05	13.0	
	F	1186	791	791	561	20LH06	14.5	
	F	1267	845	845	599	20LH07	15.6	
	F	1309	873	873	619	20LH08	16.1	
	F	1429	953	953	675	20LH09	17.2	
	F	1542	1028	1028	724	20LH10	18.0	

Figure 16: Newmill Economical Load Table for Sizing Joists

# **Electrical Breadth**

The electrical breadth was chosen in order to support the electrical load of the proposed air handling unit. First, the existing circuit was replaced with the proposed circuit, as the existing system will not be used, and the panel can handle the proposed circuit.

Using the electrical data from the submittal sheets given by H.C. Nye Co., the rating is 460V at 3 phase and the max overcurrent is 50 amps. Therefore I used a 50 amp 3 pole breaker for the circuit.

#### Sizing the Wires:

To size the wires, the minimum circuit ampacity of 42 amps was used, yielding THHW (75C) #8 AWG

#### Sizing the Groundwire:

To size the ground wire, NEC<sup>1</sup> Table 250-95 was used with a 50 amp overcurrent device, yielding **#8** AWG

#### Sizing the Conduit:

To size the conduit, PVC was chosen due to the system being outdoors, and the total wire area will be  $0.15 \text{ in}^2$ .  $\frac{3}{4}$  inch PVC conduit was chosen because the area of the conduit will be  $0.51 \text{ in}^2$ , and this will only fill 28.88% of the conduit, which is allowable by NEC of 3 or more current carrying conductors.

#### Sizing the Load on Each Wire:

The load on each wire was found using the minimum circuit ampacity.

#### 460V\*38/1000 = 17.48 kVA

Divide by 3 for each wire => 5.8266 kVA per wire



Figure 17: Proposed Panel Schedule

<sup>&</sup>lt;sup>1</sup> National Electric Code

# **Final Remarks**

#### Acknowledgments

Dr. William Bahnfleth Thesis Advisor - PSU AE Department Mr. Moses Ling Professor - PSU AE Department Mr. Daniel Scott **Construction Services - PSU OPP** Mr. Paul Moser Superintendent - Steam Services PSU OPP Mr. Bryan Ondrasik Mechanical Engineer - Buchart Horn, Inc. Mr. Justin Seltzer Project Engineer - Alexander Building Construction Co. Mr. Christopher Varughese Mechanical Engineer - MG Engineering Mr. Buck Nye President - H.C. Nye Company, Inc. Michael Connor Student - Penn State Architectural Engineering Student **Emily Blessner** Student - Penn State Architectural Engineering Student

#### **ABET Accreditation**

#### PSU AE - ABET 2.3

In order to meet ABET accreditation, the proposed systems met the following criteria. First, there was no change to the architectural design of the building. Both redesign solutions were designed with the goal of allowing the original architect's design to remain, and have very little visual impact on the building. In addition to this, one of the main needs of a building is to keep the occupants comfortable. By providing heating and cooling, both redesign proposals satisfy this need.

#### PSU AE - ABET 2.4

The proposed design solutions for this building meet the criteria of environmental, economic, and sustainability factors. Both redesign options in this project have a lower overall energy use, which is a benefit to the environment. In using less energy, the building will also have lower maintenance and upfront cost, and therefore have a lower total cost over time. Another benefit to the reduced energy use is that this yields a more sustainable and overall environmentally friendly building and building operation.

#### I. Existing Conditions

#### Monthly & Annual Totals of Energy Use

Annual Total Energy Consumption: 205,178 Btu/(SF\*year) Annual Electric Energy Consumption: 977,097 kWh Annual Steam Energy Consumption: 2,075 therms Annual Chilled Water Energy Consumption: 28,724 therms

Heating Loads

Total: 780 MBH

**Cooling Loads** 

Total: 68.3 Tons

EUI of Building: 205 kBtu/(SF\*year)

#### **Annual Energy Costs**

Total Annual Electric Consumption: \$82,402 Total Annual Chilled Water: \$52,711 Total Annual Steam: \$3,646

Utility Cost per Area: \$4.58 per SF Total Annual Utility Cost: \$143,175

#### Air Handling Unit with VAV Π.

Figure 1: Air Handling Unit Specifications



**Unit Rating** 

2425 South Yukon Ave - Tulsa, Oklahoma 74107-2728 - Ph. (918) 583-2266 Fax (918) 583-6094 AAONEcat32 Ver. 4.281 (SN: 5728272-)

### RL-045-3-0-NW0N-EHJ: ZGED-D00-KAW-000-D00AD00-00-00000000B

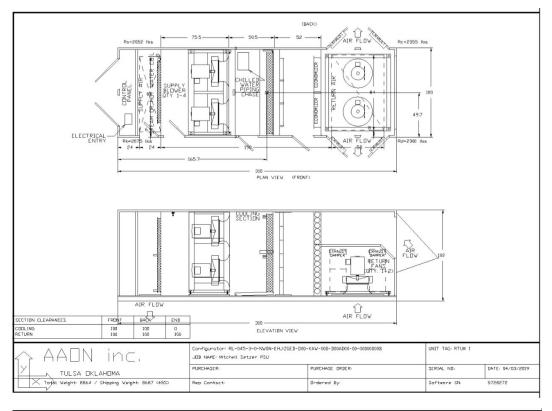
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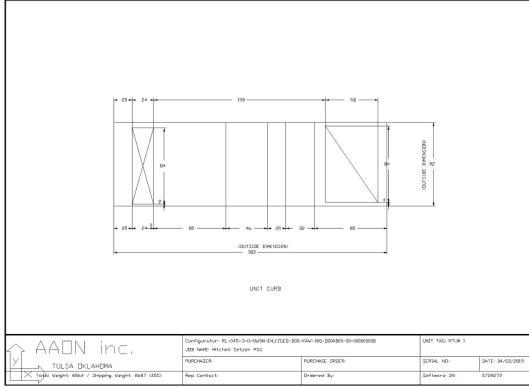
Job Information	
Job Name:	Mitchell Set
Job Number:	Job #10000
Site Altitude:	0 ft

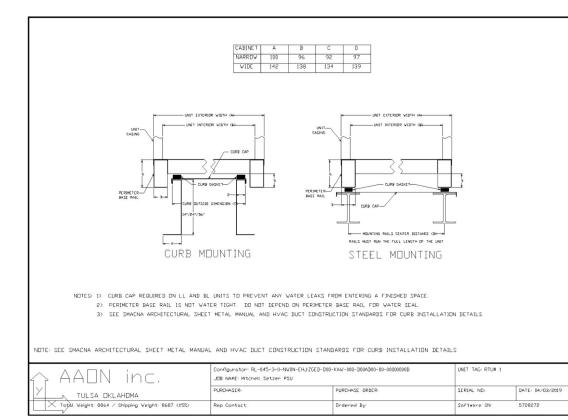
#### Unit Information

						3			
Job Name: Job Number: Site Altitude:	Mitchell Setzer PSU Job #100008739 0 ft			Approx. Op./Ship Weights: Supply CFM/ESP: Pre-Filter FV / Qty: Return CFM/ESP/TSP: Outside CFM: Ambient Temperature: Return Temperature:			8864 / 8687 lbs. (±5%) 15946 / 0.75 in. wg. 318.92 fpm / 18 15946 / 0.75 / 1.10 in. wg. 3127 86 °F DB / 74 °F WB 79.3 °F DB / 65 °F WB		
Static Pressure									
External: Coil: Filters Clean: Dirt Allowance	0.75 in. wg. 0.60 in. wg. 0.12 in. wg. 0.35 in. wg.			Econo Heatir Cabin Total:	let:	0.15 in. wg. 1.07 in. wg. 0.30 in. wg. 3.34 in. wg.			
<b>Cooling Section</b>				Heati	ing Section				
Total Capacity: Sensible Capacity: Latent Capacity: Mixed Air Temp: Entering Air Temp: Lv Air Temp (Coil): Lv Air Temp (Unit) Supply Air Fan: SA Fan RPM / Width: Return Air Fan: RA Fan RPM / Width: CW Coil: CW Face Velocity:	$\begin{array}{llllllllllllllllllllllllllllllllllll$		PreHe Heatin Total OA Te RA Te Enter Leavin GPM Water FA / F Chillle GPM	PreHeat Type: Heating Type: Heating CFM: Total Capacity: OA Temp: RA Temp: Entering Air Temp: Leaving Water: Leaving Water: Leaving Water: GPM / Head: Water Velocity: FA / RD / FPI / FV: <b>Chilled Water Coils:</b> GPM / Water PD (HXC only): Ent. / Lv. Water Temp:		Std (No Preheat) Hot Water Heat 7000 228.7 MBH 2.0 DB / 1.0°F WB 80.0 °F DB / 62.0 °F WB 64.7 °F DB / 53.9 °F WB 94.7 °F DB / 64.6 °F WB 140.0 °F 122.8 °F 27 / 2.8 ft 2.75 fps 10.63 ft* / 2 / 10 / 658.8 108 / 9.96 ft 43 / 55.6 °F 3.27 fps			
Electrical Data									
Rating: Unit FLA:	460/3/ 38	/60			num Circuit An num Overcurre		42 50		
Supply Fan: Return Fan: Control Circuit:	<b>Qty</b> 2 2 1	HP 10.00 3.00	VAC 460 460 120	Phase 3 3 1	<b>RPM</b> 1760 1170	FLA 14.0 4.8 2.9			
Cabinet Sound Power	Levels*								
Octave Bands: Discharge LW(dB): Return LW(dB):	63 92 87	125 90 87	<b>250</b> 91 82	<b>500</b> 90 78	1000 86 81	<b>2000</b> 84 77	<b>4000</b> 81 75	8000 78 74	

\*Sound power levels are given for informational purposes only. The sound levels are not guaranteed.







#### III. Electrical Breadth

#### Figure 2: Proposed Panel Board

